Amendments to the Specification:

Page 8, amend the paragraph beginning at line 11 to read as follows:

Refrigerant gas of low temperature and low pressure sucked from the suction inlet 8 provided on the motor casing 2 passes through a gas passage formed between the drive motor 7 and the motor casing 2, and an air gap defined between a stator and a motor rotor, cools the drive motor 7, and thereafter is drawn through the suction port 9 formed on the main casing 1 into a compression chamber defined by meshing tooth surfaces of the male and female screw rotors and the cylindricalshaped bore 16. As the male rotor 6m directly connected to the drive motor 7 rotates, the refrigerant gas is introduced into the compression chamber and gradually compressed as the compression chamber is reduced in volume. Thus the refrigerant gas becomes one of high temperature and high pressure to be discharged into a discharge port 14 provided in the discharge casing 3, from which discharge port the refrigerant gas passes through an oil separator inlet a discharge gas passage 20 that is formed in the discharge casing 3 and the main casing 1, and is discharged into the oil separation space 4 of the oil separator 24. Among compression reaction forces acting on the male and female screw rotors at the time of compression, a radial load is borne by the bearings 10, 11, 12 and a thrust load is borne by the ball bearing 13. Oil for lubrication and cooling of these bearings is fed due to a differential pressure through the oil feed passages 25 that are formed to be communicated with the respective bearing portions, from the oil reservoir 19 formed below a compression mechanism composed of the male and female screw rotors, and the oil thus fed is then discharged together with the compressed gas into the oil separation space 4.

Page 9, amend the paragraph beginning on line 16 to read as follows:

The oil separator inlet An inlet portion 20a of the discharge gas passage 20 is opened substantially tangentially to an inner wall of the oil separation

space 4, and a mixture of the compressed gas (refrigerant) and the oil inflows along the oil separator inner wall to go along the cylindrical-shaped inner wall to generate a swirling flow, and the oil is separated from the gas due to the centrifugal action. The oil as separated drops along the wall surface, passes through the opening 15 that provides communication between the oil separation space 4 and the oil reservoir 19 in the compressor, and is accumulated in the oil reservoir 19. By forming the opening 15 into, for example, a rectangular shape shown in Fig. 3, manufacture as by casting or the like is facilitated.

Page 11, amend the paragraph beginning on line 22 to read as follows:

The oil separator 24 is provided with a mount hole 21, to which a safety valve 22 is mounted, and a line 26 connecting between a center of the oil separator 24 and the mount hole 21 is made substantially in parallel to axes 6ma, 6fa of the screw rotors 6m, 6f. The safety valve 22 is communicated with the discharge gas passage 20 through the mount hole 21 and the oil separation space 4. With such construction, even when the safety valve 22 is mounted, a depth dimension b (see Fig. 6) is not increased, and so an installation area (a length dimension a × depth dimension b) of the compressor can be made minimum.